

NITROGEN OXIDES REDUCTION IN A POROUS BURNER

Shahrooz Afsharvahid

School of Mechanical Engineering

The University of Adelaide, South Australia 5005

A Thesis submitted in fulfilment of the requirements for the Degree of

Doctor of Philosophy

JULY 2016

STATEMENT OF ORIGINALITY

I certify that this work contains no material which has been accepted for the award of any other degree or diploma in my name in any university or other tertiary institution and, to the best of my knowledge and belief, contains no material previously published or written by another person, except where due reference has been made in the text. In addition, I certify that no part of this work will, in the future, be used in a submission in my name for any other degree or diploma in any university or other tertiary institution without the prior approval of the University of Adelaide and where applicable, any partner institution responsible for the joint award of this degree.

I give consent to this copy of my thesis, when deposited in the University Library, being made available for loan and photocopying, subject to the provisions of the Copyright Act 1968.

I also give permission for the digital version of my thesis to be made available on the web, via the University's digital research repository, the Library Search and also through web search engines, unless permission has been granted by the University to restrict access for a period of time.

ACKNOWLEDGMENTS

I would like to express my special appreciation and thanks to my one and only supervisor, *Professor Bassam Dally*. He did a remarkable job as a supervisor and mentor. I am especially thankful to his continuous encouragement, support and more importantly for his true friendship which allowed me to successfully finish this work. His advice, ideas and brilliant instructions have helped my scientific and personal development and have guided me both in my research and career life.

I would like to also thank the School of Mechanical Engineering management team (especially Professor Anthony Zander) for supporting and funding my study, the workshop and laboratory staff (especially Mr. Graham Kelly) who supported my experimental work even at times where I found it hard to keep going.

I would also like to specially thank *Professor Peter Ashman* for the great help with the few publications I had and also his remarkable knowledge of the chemical kinetics.

I like to express my special gratitude to *Professor Farid Christo* for his valuable help with the numerical modelling techniques and encouragement guided me into great world of CFD.

I need also to appreciate the patience and support from my work colleagues and the management team in *FCT Combustion*. *Constantine Manias* and *David Retallack*, are a great inspiration in terms of hardworking and going to great length to achieve desirable results. *Russell Jackson* well taught me that there is nothing wrong with being a professional and at the same time fun to work with. I found him to be an expert in transforming impossible obstacles to exciting challenges.

Special thanks go to *Ms. Alison-Jane Hunter* for her kind assistance with proofreading my thesis. Her skills and talent was instrumental in greatly improving the readability of this thesis. I appreciate her phenomenal attention to details.

Words cannot express my gratitude to my loving and caring *mother*, inspiring *father*, my two brothers, *Shahraam* and *Shahrokh*, and my beautiful newly engaged sister, *Sara*. I will always remember the huge sacrifices they made for me to give me the best possible chance in life which made me who I am today.

I would also like to thank my life friend, *Parto* for the unconditional support, love and friendship, believing in me, standing by my side and giving me the strength unconditionally. I also have been blessed to have many trustworthy and loyal friends particularly *Keivan* and *Eyad* who were always around when I needed help, encouragement or support.

Last but not least, I would like to express my deep love and gratitude to my beloved son, *Kian*. Your bright little smile has given me the motivation to keep going, from the second you opened your eyes to this world. I would like to thank you for being such a bundle of joy and laughter in my life and I feel that I am the luckiest father of all times, being able to call you my son.

ABSTRACT

Different aspects of porous burners have been studied in the past in terms of the bed material, design, heat transfer modes and flame characteristics. However, the application of porous burners to NOx reduction and the effect of the bed surface on the chemical reactions have not yet been explored. Hence, the objective of this study is to investigate the effect of the design and operating parameters on NOx reduction inside a porous burner.

To achieve this objective, a variety of flames, stabilised inside porous burners, were investigated experimentally, utilizing thermocouples, gas sampling and chromatography. Numerical tools were also used to understand the chemical pathways under different operating conditions better.

Premixed CNG-air and LPG-air flames at very low equivalence ratios were stabilised inside the porous bed. The relationship between the volumetric flow rate of the mixture and the minimum equivalence ratio was studied (experimentally and numerically) for equivalence ratios as low as ϕ =0.35 (equivalent to thermal power of 2kW). The maximum temperature observed to be consistent with super-adiabatic flame temperatures. The maximum measured NO_X and CO mole fractions at the burner exit were found to be in the order of few PPMs.

The conversion of NOx was then assessed. A mixture of CNG-air doped with NO was introduced into the burner inlet and the effects of the operating parameters on NOx reduction were assessed. It was found that NOx reduction is a function of the equivalence ratio, total flow rate and NO mole fraction at the inlet. Higher flow rates led to an increase in the conversion rate at higher equivalence ratios, due to shorter residence times, and the greater need for more flame radicals in the flame.

The numerical study revealed that different chemical pathways dominate at different equivalence ratios, which led to the production of other intermediates and stable radicals. The study showed that the Total Fixed Nitrogen, TFN, reduction followed a similar trend to the NOx reduction for moderately fuel-rich conditions ($\phi \leq 1.2$) and opposite trends for higher equivalence ratios. For $\phi>1.2$, most of the NO is converted to N-containing species such as N₂O, NH₃ and HCN and not to N₂. Analysis of the chemical pathways showed that the formation of nitrogen-containing species under very fuel rich conditions is due to the increased importance of the HCNO path, as compared with the HNO path. The best TFN conversion efficiency, 65%, was found at $\phi=1.1$.

Intermediate radicals have different rates of destruction and production on the porous bed surface, especially for mixtures close to stoichiometric conditions. Under these conditions, the conversion of NO_X is strongly influenced by the concentration of H radicals. A collision probability of $\eta = 8 \times 10^{-4}$ was found to represent this radical loss effect and to help predict the destruction and production of intermediate terminals with a good level of accuracy.

This study also found that NOx reductions using porous burners are technically feasible and that the resulting CO in the exhaust, derived from the rich mixtures, can be burned outside the porous bed.

TABLE OF CONTENTS

Statemen	nt of Originalityii
Acknowl	ledgmentsiii
Abstract	iv
Table of	contentsv
List of T	ablesx
List of F	iguresxi
Nomenc	laturexviii
1. Intr	roduction1
1.1	Importance of combustion 1
1.2	Environmental pollution
1.3	Alternative combustion systems 4
1.4	Porous Media Combustion
1.4.1	Advantages of porous burners
1.4.2	2 Applications of porous burners
1.4.3	3 Limitations of porous burners
1.5	Motivation for the research9
1.6	Scope
1.7	Thesis structure
2. Bac	kground
2.1	Porous Burners: Principles of operation
2.2	Porous medium materials and shapes
2.2.1	Porous foams/fibres
2.2.2	2 Metallic wire meshes and foils
2.2.3	B Discrete materials
2.3	Flow inside porous medium
2.4	Combustion within porous media

	2.4.1	Ign	ition, heat-up process and stabilisation in PB	20
	2.4.2	Pro	pagation speed	21
ź	2.5	Super	r-adiabatic flame temperature (excess enthalpy combustion)	25
,	2.6	Multi	stage combustion in porous burners	26
,	2.7	Liqui	d fuels	27
2	2.8	Pollut	tant emissions	29
2	2.9	NO _X i	reduction mechanisms	32
,	2.10	Surfa	ce reactions	35
,	2.11	Mode	elling	36
,	2.12	Resea	arch Objectives and Gap	39
<i>3</i> .	Bed	Fuel	Injection in a Porous Burner	40
,	3.1	Intro	duction	40
,	3.2	Expe	rimental setup	41
	3.2.1	Swi	irl-burner	41
	3.2.2	Hea	at exchanger	43
	3.2.3	Por	ous burner	44
	3.2	2.3.1	Main tube	44
	3.2	2.3.2	Insulation	45
	3.2	2.3.3	Temperature measurements	46
	3.2	2.3.4	Material selection	46
	3.2.4	Fue	l distribution system	47
	3.2.5	Cor	ntrol system and Data collection	47
	3.2	2.5.1	Single-tube fuel distributor	49
	3.2	2.5.2	Spiral-tube fuel distributor	50
	3.2	2.5.3	Multi-tube fuel distributor	51
,	3.3	Resul	ts and discussion	52
	3.3.1	Sin	gle-tube fuel distributor	53
	3.3.2	Spi	ral-tube fuel distributor	55
	3.3.3	Mu	lti-tube fuel distributor	59
ć	3.4	Sumn	nary and Conclusions	64
4.	Pren	nixed	Air/Fuel in a Porous Burner	65
	<i>1</i> 1	Intro	duction	65

	4.2	Experimental setup	66
	4.2.1	Electrical heater	66
	4.2.2	Porous medium and temperature measurement	67
	4.2.3	Start-up	68
	4.3	Numerical modelling	68
	4.3.1	Model description	69
	4.4	Results and discussion	69
	4.4.1	Validating of the 1-D assumption	69
	4.4.2	Heating up process and propagation speed	70
	4.4.3	Effect of flow velocity on flame stabilisation and the location of the flame	73
	4.4.4	The effect of equivalence ratios on flame front locations	79
	4.4.5	Effect of fuel type on the flame front location	81
	4.4.6	Super-adiabatic flame temperature and excess enthalpy	83
	4.4.7	Pollutants Emission	84
	4.5	Summary and Conclusions	86
5.	Porc	ous Burner as a Post-Combustion medium	88
	5.1	Introduction	88
	5.2	Experimental setup	90
	5.2.1	Heat exchanger	93
	5.2.2	Control systems and measuring devices	94
	5.3	Numerical modelling	95
	5.4	Results and Discussion	96
	5.4.1	Flame temperature and heat loss	96
	5.4.2	CO emission	100
	5.4.3	Effect of Equivalence Ratios on NOx and TFN	101
	5.4.4	Effects of Flow Velocity (residence time) on NOx and TFN	108
	5.4.5	Effect of Initial NO _X Mole Fraction	111
	5.4.6	Effect of CO ₂ in the inlet mixture on NOx and TFN	112
	5.5	Summary and Conclusions	113
6.	Surf	ace Reaction in Porous Burners	115
	6.1	Introduction	115
	6.2	Numerical Modelling	116

6.4	Summary and Conclusions	138
7. Su	mmary, Conclusions and Future Work	140
7.1	Porous Burner as a Combustion Medium	141
7.1.	.1 The Effect of Fuel Mixing	141
,	7.1.1.1 Porous Burner with Bed Fuel Injection	141
ŕ	7.1.1.2 Porous Burner with premixed Fuel Injection	142
7.2	Porous Burner as a Post Process Combustion Medium	142
7.2.	.1 NOx reduction in Porous Media	143
,	7.2.1.1 Effects of equivalence ratio on NOx conversion efficiency.	144
,	7.2.1.2 Effects of Flow Velocity on NOx conversion efficiency	145
,	7.2.1.3 Effects of Input NO on NOx conversion efficiency	145
7.2	.2 TFN Reduction in Porous Burners	145
7.2.	Effect of Surface Reaction on NOx/TFN Reduction	146
7.3	Conclusions	146
7.4	Future Work	149
7.4	.1 Porous Bed Material and Geometry	149
7.4	2.2 Effects of actual exhaust gases on TFN conversion	150
Referen	ıces	
Append	lices	165
A.]	Publications originating from this study	165
В.	Calculating holes distances in multi-tube fuel distributor	167
C. 1	Modelling porous burner using PBM	168
C.1	Governing equations	168
C.1.1	Mass continuity equation	168
C.1.2	Gas species conservation equation	168
C.1.3	Gas-phase energy equation	168
C.1.4	Solid-phase energy equation	170
C.1.5	Heat convection coefficient	170
C.2	2 Radiation model	171
C.3	Boundary conditions	171
C.4	Numerical solver	172

D.	Appendix C: Modified GRI-Mech 3.0 Chemica	l Kinetic Mechanism Used in
Surf	face Reaction Calculations	175
E.	Conversion of Selected Flow Rates and Equiva	lence Ratios to Flow Velocities
and	Power	
E.	.1 CNG/Air Mixtures	
E.	2 LPG/Air Mixtures	

LIST OF TABLES

Table 2.1: Steady-state regimes and reaction transfer mechanisms for gas combustion in inert porous
media [60, 61]
Table 3.1: Technical specification of gas analyser as per calibration certificates
Table 3.2: CNG composition and physical properties
Table 4.1: LPG composition and physical properties
Table 4.2: Comparison of equilibrium and measured flame temperatures and relevant measured NO_X
and CO mole fractions, in the exhaust, for CNG/air flames
Table 4.3: Comparison of equilibrium and measured flame temperatures and relevant measured NO_X
and CO mole fractions, in the exhaust, for LPG/air flames
Table 6.1: Arrhenius parameters for reactions added to GRI-Mech 3.0 and Konnov mechanisms that
account for surface reactions
Table 6.2: Net average reaction rates (1 $ imes$ 10 10) of reactions affecting NO reduction for cases 1 to 8.
Table 6.3: Branching ratios of NH between N_2O , N_2 , NO and NH_3 compared with the total NH
produced
Table 7.1: Conversion of selected flow rates and equivalence ratios to flow velocities and power for
CNG/air mixtures
Table 7.2: Conversion of selected flow rates and equivalence ratios to flow velocities and power for
LPG/air mixtures

LIST OF FIGURES

Figure 1.1: Porous burner schematic (left), Regular porous burner (middle) [30], and Porous radiant
burner (right) [31]
Figure 1.2: Heat and mass transfer in a schematic porous medium, formed by a continuous gas phase
and dispersed solid phase [56] 9
Figure 2.1: Schematic of a porous burner and its principles of operation
Figure 2.2: Different ceramic porous foams: (a) Al_2O_3 fibres, (b) C/SiC^{TM} structure and (c) static mixer
made of ZrO ₂ foams [32]
Figure 2.3: Fe-Cr-Al-alloy wire mesh[32]
Figure 2.4: various combustion regimes as a function of excess air E_a and firing rate Q for three
different types of burners: non-catalytic, surface catalysed and the fully catalysed structures [28] 21
Figure 2.5: Stabilisation diagram for flames stabilised inside and outside a porous burner in ambient
temperatures and in hot environments (T_{env} = 750 K, dashed line, T_{env} =775 K, dotted line) for Φ = 0.9 [74].
Figure 2.6: Stability diagram with respect to the lower flammability limit and the matrix porosity for
three distinct values of firing rate [75]
Figure 2.7: Stability diagram with respect to flammability limits and turn–down ratio [75]23
Figure 2.8: NO_X and CO emissions of a 30 kW porous media burner in comparison with stringent
European standards [25]
Figure 2.9: NO_X emissions as a function of excess air E_a for different firing rates Q for three ceramic
burners: non-catalytic, surface-catalysed and the fully-catalysed burners [28]
Figure 2.10: CO emissions as a function of excess air Ea for two different firing rates Q=190 and
300kW/m2 for the three ceramic burners: non-catalytic, surface-catalysed and the fully-catalysed
burners [28]
Figure 3.1: Schematic description of swirl burner
Figure 3.2: Manufactured swirl burner, air and fuel inlets (left), bluff body and flow straightener
(right)
Figure 3.3: Manufactured (left) and schematic view of heat exchanger (Right)44
Figure 3.4: A view of the manufactured (left) and designed (right) ceramic tube
Figure 3.5: Varity of material used as packed bed: (a) Calcinated flint clay, (b) Alumina ceramic beads
and (c) alumina saddles as an alternative for ceramic beads
Figure 3.6: Gas analyser system showing the different analysers and data acquisition system 48
Figure 3.7: Different fuel distributor designs used in the experimental study: (a) single-tube, (b) spiral-
tube and (c) multi-tubes

Figure 3.8: The porous burner assembly: schematically (right), and as assembled in the laboratory	
(left)	50
Figure 3.9: Leister electrical air heater 10000 S used in the 'porous bed fuel injection' experiments	51
Figure 3.10: Porous burner assembly with electrical heater	52
Figure 3.11: Temperature profiles along the centreline of the ceramic tube. \Box Initial position of the	
flame in the bed; ♦ final position of the flame in the bed. A single-tube is used as fuel distributor	54
Figure 3.12: Top view picture of the PB using a single-tube fuel distributor. Schematic view and fuel	
distributor insertion (left)	54
Figure 3.13: The radial flame temperature gradient in a single fuel distributor. Measurements were	
from a +125mm axial location of the porous bed	55
Figure 3.14: Temperature profiles along the centreline of the ceramic tube when a spiral-tube is use	d
as a fuel distributor. ◆, □ and ▲ show a fuel/air mixture with a total flow rate of 265 lit/min and GE	ĒR
of 0.4, 0.5 and 0.7 respectively	56
Figure 3.15: Temperature profiles along the centreline of the ceramic tube for different GERs. $lacktriangle$, \Box	
and ♦ show a global equivalence ratio of 0.15, 0.2, and 0.3 and a flow rate of 200, 150 and	
$100 lit/min\ respectively.$ The firing rate is fixed at $92 kW/m^2$ for all cases. A spiral-tube is used as a full case of the spiral case of th	el
distributor	57
Figure 3.16: Top view pictures of PB using spiral-tube fuel distributor: (a) igniting on the top surface	,
(b) glowing and start propagating and (c) propagation upstream and stabilisation	58
Figure 3.17: Testing of gas distribution uniformity in a spiral-tube fuel distribution system. Oxygen	
$was \ used \ (no\ combustion)\ in\ the\ fuel\ distributor\ and\ the\ concentration\ of\ oxygen\ was\ measured\ at$	
different radial locations at 125mm downstream of the fuel distributor	59
Figure 3.18: Measured temperature profiles along the centreline of the ceramic tube for flow rates of	of
300 (st), 400 ($lacktriangle$), 500 (\Box) and 600 ($lacktriangle$) lit/min and a global equivalence ratio of $\overline{\Phi}$ =0.3	60
Figure 3.19: Measured temperature profiles along the centreline of the ceramic tube for global	
equivalence ratio of flow rates of $\overline{\Phi}$ = 0.3 (\spadesuit), $\overline{\Phi}$ = 0.275 (\square), $\overline{\Phi}$ = 0.25 (\blacktriangle), $\overline{\Phi}$ = 0.225 ($*$), $\overline{\Phi}$ = 0.2	
$(+),\overline{\Phi}$ = 0.175 (\triangle) and $\overline{\Phi}$ = 0.15 $(ullet)$ and a flow rate of 500 lit/min. A multi-tube is used as a fuel	
distributor	61
Figure 3.20: Comparison of experimental and equilibrium temperatures and measured NO $_{ exttt{X}}$ for ($\overline{\Phi}$ =	=
0.15 to $\overline{\Phi}$ = 0.3) and atotal flow rate of 500 lit/min. $lacktriangle$ and \Box adiabatic flame temperatures and	
measured flame temperatures respectively. ▲ and ● measured NO _x concentrations (absolute value	es
and at 3% oxygen respectively. A multi-tube fuel distributor is used	62
Figure 3.21: Top view of the PB using a multi-tube fuel distributor: (a) igniting on the top surface, (b))
glowing and starting to propagate and (c) propagation upstream and stabilisation inside the porous	5
medium	63

Figure 3.22: Fuel distribution uniformity in a multi-tube fuel distribution system. Oxygen was used (no
combustion) in the fuel distributor and the concentration of oxygen was measured in different radial
locations at 225 mm downstream of the fuel distributor
Figure 4.1: Assembly for feeding premixed air/fuel (left), a close view of the air and fuel supply lines
(right)
Figure 4.2: Leister electric air heater 1000 S (left), Air heater assembled upstream of the porous
medium (right)
Figure 4.3: Schematic description of the computational domain for the porous burner model 69
Figure 4.4: Measured Temperature radial profiles at different axial locations for LPG/Air flames with
φ = 0.39
Figure 4.5: Measured centerline temperatures at different axial locations inside the porous bed
during the heat up process, using an external swirl-burner71
Figure 4.6: Measured centerline temperatures at different axial locations inside the porous bed
during the heat up process, using an electrical heater
Figure 4.7: Measured centerline temperatures at different axial locations inside the porous bed
during the heating up process, using combustion in the porous bed (recuperating)73
Figure 4.8: Measured centreline temperature profiles for CNG/air flame at ϕ = 0.4 and for different
flow velocities
Figure 4.9: Measured centreline temperature profiles for LPG/air flame at ϕ =0.4 and different flow
velocities
Figure 4.10: Comparison of measured and calculated centerline temperatures for initial velocities of 9
cm/sec and 19 cm/sec and an equivalence ratio of φ = 0.35
Figure 4.11: : Comparison of measured and calculated centerline temperatures for initial velocities of
9 cm/sec and 19 cm/sec and an equivalence ratio of ϕ = 0.40
Figure 4.12: Comparison of calculated flame temperatures for cases with inlet flow velocities of 19
cm/s, 38 cm/sec and 56 cm/sec and an equivalence ratio of φ = 0.40
Figure 4.13: Temperature profiles for an inlet flow velocity of 19 cm/sec and different equivalence
ratios for CNG/air flames
Figure 4.14: Temperature profiles for an inlet flow velocity of 19 cm/sec and different equivalence
ratios for LPG/air flames
Figure 4.15: Comparison of measured and calculated centerline temperatures for a flame with an
inlet flow velocity of 19cm/sec and for φ = 0.35 (a) and φ = 0.40 (b). The solid lines are the calculated
bed temperature; the dashed lines are the calculated gas temperature and the symbols are the
measured gas temperature
Figure 4.16: Comparison of CNG/air and LPG/air flames for similar equivalence ratios and different
flow velocities

Figure 4.17: Comparison of CNG/air and LPG/air flames for similar equivalence ratios and flow
velocities
Figure 4.18: Comparison of measured and calcuated maximum flame temperatures for CNG flames
with equivalnce ratios of ϕ = 0.35 and ϕ = 0.4. Also plotted are the equilibrium flame temperatures
for two equivalence ratios84
Figure 4.19: Comparison of measured maximum flame temperature for LPG flames with equivalnce
raios of ϕ = 0.35 and ϕ = 0.4. Also plotted are the equilibrium flame temperature for two equivalence
ratios
Figure 5.1: New porous burner setup and schematic
Figure 5.2: Sketch of the heat exchanger design94
Figure 5.3: Photograph of the heat exchanger assembly94
Figure 5.4: Measured maximum flame temperatures in the porous burner, along with the adiabatic
flame temperature for fixed air flow rates of 150 slpm versus the equivalence ratios. The solid line
represents the adiabatic flame temperature, (Solid line, ♦) represents the measured maximum
temperature and (dashed line, $ullet$) represents the ratio of heat extracted by the heat exchanger to the
firing rate
Figure 5.5: Temperature versus equivalence ratios for a flow velocity of 25 cm/sec (left) from another
porous burner using Methane as fuel [51]. Propagation wave velocity plotted versus different
equivalence ratios (right) [51]
Figure 5.6: 3D contour presenting the measured centreline axial temperature for different
equivalence ratios and a fixed inlet air flow rate of 100 slpm
Figure 5.7: Measured radial temperature profiles for an air flow rate of 50 slpm and equivalence ratio
of 1.5. (Solid line, ♦) and (solid line, ■) show radial temperatures at 85 mm and 135mm above the
heat exchanger, respectively
Figure 5.8: Calculated CO mole fractions at the exit of the porous burner for an air flow rate of 100
slpm and input NO of 100 ml plotted versus the equivbalence ratio
Figure 5.9: Measured NO_X conversion ratios and calculated NO_X and TFN conversion ratios plotted
versus the equivalence ratio for an air flow rate of 100 slpm and initial NO level of 100 ml. The black
solid line, grey solid line and (dashed line, \blacksquare) represent the calculated TFN conversion, calculated NO $_X$
conversion and experimental NO_X conversion, respectively. The dotted line shows the calculated
residence time for different equivalence ratios
Figure 5.10: Axial temperature profiles for different equivalence ratios and inlet air flow rates of 100
slpm (left). Zoomed-in profiles close to the flame front (right). (Dashed line, ●), (solid line, ■) and
(Dotted line A) represent equivalence ratios of 1.1.1.5 and 1.9 respectively 103

Figure 5.11: Calculated mole frcation of major N containing species for an air flow rate of 100 slpm,
an equivalence ratio of 1.1 and 100 ml of NO at the inlet. Note the different scales in upper and lower
graphs
Figure 5.12: Mechanism of NO reduction for an air flow rate of 100 slpm, an equivalence ratio of 1.1,
and 100 ml of input NO. The thickness of the arrows is indicative only and is not scaled accurately.
Figure 5.13: Calculated mole fractrions of major N containing species for an air flow rate of 100 slpm,
an equivalence ratio of φ = 1.7 and 100 ml of input NO. Note the different scales in upper and lower
graphs
Figure 5.14: Mechanism of NO reduction for an air flow rate of 100 slpm, an φ = 1.7, and 100 ml of
input NO. Note that the arrows' thicknesses are indicative and are not scaled accurately 107
Figure 5.15: Measured axial temperature profiles for different air flow rates and for an equivalence
ratio of 1.1. (Dotted line, \blacksquare), (Dashed line, \spadesuit), (solid line, \blacksquare) and (Dashed-Dotted line, \blacktriangle) represent air
flow rates of 50 slpm, 100 slpm, 150 slpm and 200 slpm, respectively
Figure 5.16: Measured and calculated NO_X conversion ratios plotted versus air flow rates for an input
NO level of 100 ml and φ = 1.1. Black solid line, grey solid line and (dashed line, \blacksquare) represent the
calculated TFN conversion, calculated NO $_{\rm X}$ conversion and experimental NO $_{\rm X}$ conversion, respectively.
The dotted line represents the residence time for different cases
Figure 5.17: Measured and calculated NO_X conversion ratios plotted versus the input NO levels for air
flow rates of 100 slpm and φ = 1.1. The black solid line, grey solid line and (dashed line, \blacksquare) represent
the calculated TFN conversion, calculated NO $_{\rm X}$ conversion and experimental NO $_{\rm X}$ conversion,
respectively
Figure 5.18: Measured and calculated NO $_{\rm X}$ conversion ratios plotted versus the input NO levels for air
flow rates of 100 slpm, and φ = 1.7. The black solid line, grey solid line and (dashed line, \blacksquare) represent
the calculated TFN conversion, calculated NO_X conversion and experimental NO_X conversion,
respectively
Figure 5.19: The effect of CO_2 addition on the measured NO_X conversion efficiency at different
equivalence ratios, for 100slpm and an input NO of 100 ml
Figure 6.1: Measured NO _x conversion ratios and calculated NO _x plotted versus equivalence ratio for
an air flow rate of 100 slpm and initial NO level of 100 ml. Lines (from top to bottom) represent η = 0,
η = 4×10 ⁻⁴ , η = 6×10 ⁻⁴ , η = 8×10 ⁻⁴ , η = 10×10 ⁻⁴ and η = 1 (100% collision probability), respectively.
Also, ▲ represents the experimental NO _x conversion efficiency
Figure 6.2: Measured NOx conversion ratio and calculated NOx (using Konnov mechanism) plotted
versus the equivalence ratio for an air flow rate of 100slpm and initial NO level of 100ml. Lines (from
top to bottom) represent $\eta = 0$ and $\eta = 8 \times 10^{-4}$. Also, \blacktriangle represents experimental NO _x conversion
efficiency

Figure 6.3: Comparison of N-containing species calculated using GRI-Mech 3.0. Left figure show
predictions for no surface reactions (η = 0) and the right plot show predictions with surface reactions
for η = 8×10 ⁻⁴ . All predictions are for flow rates of 100 slpm, φ = 1.1 and input NO of 100 ml 123
Figure 6.4: Comparison of N-containing species calculated using Konnov mechanism; Left plot show
predictions for no surface reactions (η = 0) and the plot in the right show predictions with surface
reactions for η = 8×10 ⁻⁴ . All predictions are for flow rates of 100 slpm, φ =1.1 and input NO of 100 ml.
Figure 6.5: Integrated chemical path for a flow rate of 100 slpm using GRI-Mech 3.0, ϕ = 1.1, input
NO of 100ml and η = 0. Arrow thicknesses are scaled based on the reaction rates for different
reactions
Figure 6.6: Integrated chemical path for a flow rate of 100 slpm using GRI-Mech 3.0, ϕ = 1.1, input
NO of 100ml and η = 8×10^4 . Arrow thicknesses are scaled based on the reaction rates for different
reactions
Figure 6.7: Integrated chemical path for a flow rate of 100 slpm using the Konnov mechanism, ϕ =
1.1, input NO of 100 ml and η = 0. Arrow thicknesses are scaled based on the reaction rates for
different reactions
Figure 6.8: Integrated chemical path for a flow rate of 100 slpm using the Konnov mechanism, ϕ =
1.1, input NO of 100ml and $\eta = 8 \times 10^4$. Arrow thicknesses are scaled based on the reaction rates for
different reactions
Figure 6.9: Measured NOx conversion ratio and predicted NOx (using GRI-Mech 3.0 mechanism)
plotted versus the equivalence ratio for an air flow rate of 100 slpm and initial NO level of 100 ml.
Data and predictions are shown for ϕ = 1.0 to ϕ = 1.3. Lines (as labeled) show the model predictions
using the GRI-Mech 3.0 mechanism for the following cases: (i) No surface reactions (η = 0); (ii)
Surface reactions ($\eta = 8 \times 10^{-4}$); (iii) No surface reactions, A_{212} decreased by a factor of 4.0 and A_{214}
decreased by a factor 2.0; and (iv) No surface reactions, A_{212} decreased by a factor of 1.4 and A_{214}
increased by a factor of 2.0
Figure 6.10: Arrhenius plot of recommended rate constants for reaction R212: $H + NO + M \rightarrow HNO +$
M (M = N 2). The vertical dashed lines indicate the approximate temperature range of interest for this
work
Figure 6.11: Arrhenius plot of recommended rate constants for reaction R214: HNO + H \rightarrow H2 + NO in
the temperature range 1000 - 1600 K
Figure 6.12: A comparison of N-containing components; no surface reactions (left) and applying
surface reactions with η = 8e-4 (right). Both graphs are plotted based on a flow rate of 100 slpm, ϕ =
1.7 and input NO of 100 ml
Figure 6.13: Integrated chemical paths for a flow rate of 100 slpm, η =8×10 ⁻⁴ , an equivalence ratio of
1.7 and input NO of 100 ml

Figure 6.14: Measured NO _x conversion ratios and calculated NO _x plotted versus input NO for an air
flow rate of 100 slpm and an equivalence ratio of ϕ = 1.1. Lines (from top to bottom) represent η = 0,
$\eta = 4 \times 10^{-4}, \ \eta = 6 \times 10^{-4}, \ \eta = 8 \times 10^{-4}, \ \eta = 10 \times 10^{-4} \ and \ \eta = 1, \ respectively.$
Figure 6.15: Comparison of normalised N-containing components for an air flow rate of 100 slpm,
equivalence ratio of 1.1 and input NO of 20 ml (left) and 100 ml (right) considering η = 8×10 4 . Note
the different scales in the upper and lower graphs
Figure 6.16: Measured NO_X conversion efficiency and calculated NO_X plotted versus the air flow rate
for an equivalence ratio of 1.1 and initial NO level of 100ml. Lines (from top to bottom) represent η =
0, $\eta = 4 \times 10^{-4}$, $\eta = 6 \times 10^{-4}$, $\eta = 8 \times 10^{-4}$, $\eta = 10 \times 10^{-4}$ and $\eta = 1$, respectively
Figure 6.17: Measured axial temperature profiles for different air flow rates and for an equivalence
ratio of 1.1. (Dotted line, \blacksquare), (Dashed line, \spadesuit), (solid line, \bullet) and (Dashed-Dotted line, \blacktriangle) represent
air flow rates of 50 slpm, 100 slpm, 150 slpm, and 200 slpm, respectively.

NOMENCLATURE

i. Roman Symbols

Symbol	Definition	Unit
а	Surface Area Density	m ⁻¹
\boldsymbol{A}	Arrhenius pre-exponential factor	s^{-1}
A_{up}	Burner Cross Sectional Area	M^2
C_m	Measured NO _x mole fraction	PPM
Cp	Specific Heat Capacity	kJ.K ⁻¹
c_{pg}	Specific heat of gaseous species	kJ.kg ⁻¹ .K ⁻¹
C_r	Reference NO _x mole fraction	PPM
$\varDelta {G_{ki}}^*$	The Gibbs free energy	$Kg.m^2.s^{-2}$
d_h	Hydraulic Diameter	M
D_i	Molecular diffusivity of the <i>i</i> th species in Nitrogen	$m^2.s^{-1}$
d_p	Pore Diameter	M
\boldsymbol{E}	Arrhenius activation energy	J.mol ⁻¹
EI_i	Emission Index	-
Eli	Emission Index	-
F	Inertia coefficient	m^{-1}
f	Flame Location	m
h_{V}	Convective heat transfer coefficient for the porous medium	W.m ⁻² .K ⁻¹
K	Permeability	m^2
K	Specific permeability of the porous medium	m^2/kg
k_{cj}	Diffusion rate of radical species, i, to the burner surface	s^{-1}
$k_{coll,i}$	Surface collision rate constant for species i	s^{-1}
$k_{e\!f\!f}$	Effective rate of radical termination at the burner surface	s^{-1}
$k_{g,e}$	Effective thermal conductivity of the gas	W.m ⁻¹ .K ⁻¹
k_j	First-order reaction rate constant for the j^{th} reaction	s^{-1}
kk	Total number of gaseous species	-
L	Bed length	m

Nomenclature

m	Measured Oxygen concentration	Mole.m ⁻³
$m^{\prime\prime}$	Mass flow rate per unit area	kg.m ⁻² .s ⁻¹
M_i	Molecular weight of the <i>i</i> th species	g.mol ⁻¹
Nu_d	Nusselt number based on the average particle diameter of the packed bed	d -
P	Pressure	Pa
Pe	Péclet number	-
Pr	Prandtl number	-
R	Universal gas constant, 8.314	J/mol/K
R	Reference Oxygen concentration	-
Re	Reynolds number	-
S_p	Laminar Flame Speed	m.s ⁻¹
T	Temperature	K
T_{amb}	Ambient temperature	K
T_g	Gas temperature	K
T_s	Solid temperature	K
u_p	Superficial velocity (cross sectional velocity)	m.s ⁻¹
V_{bed}	Cross-sectional mean velocity (Darcian velocity)	m.s ⁻¹
V_p	Total Volume of Pebbles	m^3
V_{pm}	Volume of Porous Media	m^3
\overline{C}_i	Mean gas speed of species i	m.s ⁻¹

ii. Greek Symbols

Symbol	Definition	Unit
Φ	Equivalence ratio	-
γ	Radical recombination efficiency = k_{eff} / k_{coll}	-
η	Relative rate of radical termination = k_{eff}/k_{ci}	-
μ	Dynamic viscosity	kg.s.m ⁻¹
n	Arrhenius temperature coefficient	-
heta	Burner surface-to-volume ratio (6400m ⁻¹ , for this burner)	m^{-1}
ρ	Gas density	Kg.m ⁻³
σ	Stefan-Boltzmann constant	W.m.K ⁻⁴
$\overline{\sigma}$	Average Reaction Rate	Mol.s ⁻¹
Δ	Packed Bed Sphere Diameter	m
Е	Porosity	-
٨	Coefficient of Thermal Conductivity	-
v	Kinematic viscosity	$m^2.s^{-1}$
σ(x)	Net Reaction Rate	Mol/cm ³ s
$ au_{e\!f\!f}$	Effective Residence Time	S
$\overline{\Phi}$	GER, Global equivalence ratio	-

iii. Acronyms and Abbreviations

Acronym	Definition
CNG	Compressed Natural Gas
FLOX	Flameless Oxidation
HVR	High Velocity Regime
LPG	Liquefied Petroleum Gas
LVD	Low Velocity Detonation
LVR	Low Velocity Regime
MILD	Moderate or Intense Low oxygen Dilution
ND	Normal Detonation
NOx	Nitrogen Oxides
PB	Porous Burner
PBM	Porous Burner Model
PPB	Part Per Billion
PPM	Part Per Million
PRB	Porous Radiant Burner
RCR	Rapid Combustion Regime
SCW	Super-adiabatic Combustion Wave
SVR	Sound Velocity Regime
TFN	Total Fixed Nitrogen