

Faculty of Engineering, Computer and Mathematical Sciences school of mechanical engineering

# Modelling and design of magnetic levitation systems for vibration isolation

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## Acknowledgements & Dedication

Thanks to my supervisors Ben and Anthony, who gave me the freedom to be myself and who inspired me beyond my work. My appreciation couldn't be greater for the long leash they allowed me. With stricter guidance I may have finished sooner, but I believe neither I nor this thesis would have been the better for it.

This thesis is dedicated to those many with whom I've shared this happy life, especially to those in this time who have come into this strange world and to those who have departed from it.

#### Abstract

Vibration disturbance has a consistent negative impact on equipment and processes. The central theme of this thesis is the investigation of using permanent magnets in the design of a system for vibration isolation.

The thesis begins with a comprehensive literature review on the subjects of passive and active vibration isolation, permanent magnetic systems, and the common area between these on nonlinear vibration systems using magnetic forces. The use of cylindrical and cuboid magnets is the primary focus of this work for which analytical solutions are known for calculating forces and torques. Subsequently, the state of the art in analytical modelling of permanent magnet systems is covered, including a contribution in this area for calculating the forces between cylindrical magnets.

A range of load bearing designs using simple permanent magnet arrangements are examined, with multiple designs suitable for a variety of objectives. A particular emphasis is placed on a system using inclined magnets, which can exhibit a loadindependent resonance frequency. Load bearing using multipole magnet arrays is also discussed, in which a large number of magnets are used to generate more complex magnetic fields. A variety of multipole arrays are compared against each other, including linear and planar magnetisation patterns, and an optimisation is performed on a linear array with some resulting guidelines for designing such systems for load bearing.

Permanent magnet levitation requires either passive or active stabilisation; therefore, the design of electromagnetic actuators for active control is covered with a new efficient method for calculating the forces between a cylindrical magnet and a solenoid. The optimisation of a solenoid actuator is performed and geometric parameters are found which are near-optimal for a range of operating conditions.

Two quasi-zero stiffness systems are introduced and analysed next. These systems are designed with a nonlinearity such that low stiffnesses are achieved while bearing large loads. The first system analysed is a purely mechanical device using linear springs; unlike most analyses of this design, the horizontal forces are also considered and it is shown that quasi-zero stiffness is capable in all translational directions simultaneously. However, a notable disadvantage of such spring systems is their difficulty in online tuning to adapt to changing operating conditions. A magnetic quasi-zero stiffness system is then analysed in detail and design criteria are introduced, providing a design framework for such systems and showing how the complex interaction of variables affects the resulting dynamic behaviour. Although the system is nonlinear, the effects of the nonlinearities on the vibration response are shown to be generally negligible.

The thesis concludes with some experimental results of the same quasi-zero stiffness system, constructed as a single degree of freedom prototype. The quasistatic and dynamic behaviour of the system matches the theory well, and active vibration control is performed to improve the vibration isolation characteristics of the device.

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## Nomenclature

Most mathematical symbols used in this document are listed herein with page number cross-referencing. Page numbers listed will not include where these symbols are used in graphs or some figures. There are some cases where the same symbol has been used for multiple purposes; these should be unambiguous based on the context of the work.

In mathematical contexts, care has been take to be consistent in the use of parentheses for function arguments (such as  $f(x) = x^2$ ) and square brackets for both grouping  $(A \times [B \times C])$  and vector/matrix notation ( $\mathbf{F} = [F_x, F_y, F_z]^T$ ).

Symbol	Description	Page
A	Vector area (direction normal to plane)	48, 52
Α	Area	58, 59
$A_e$	Amplitude of sinusoidal base excitation	40, 208–211
а	Half-length of cuboid magnet, $\hat{x}$ direction	60, 62, 65, 67, 68
а	Cuboid magnet depth	80, 84, 109, 110
а	Cube magnet side length	xii, xiii, 64, 65,
		192–194, 196–209
$a_w$	Cross-sectional area of the wire in the coil	166
В	Magnetic flux density vector	28, 33, 48–52, 57, 59, 158
$B_r$	Remanence magnetisation	50–52, 55, 58, 61–68,
	Ŭ	73, 75, 76, 120, 144,
		146, 155, 157–159,
		161, 164, 167,
		195–197, 215
b	Cuboid magnet face side length	80, 84, 108–110,
		133–136, 138, 139,
		141
b	Half-length of cuboid magnet, $\hat{y}$ direction	60, 62, 65, 67–69
$[BH]_{max}$	Maximum energy product	51, 55
$C_i$	Compression of the inclined springs at quasi-zero stiffness	189
$C_v$	Compression of the vertical spring at quasi-zero stiffness	189
С	Relative velocity damping coefficient	xxiii, 5, 7, 8, 16, 119,
C	Relative velocity damping coefficient	120
С	Damping coefficient	5, 7–10, 18, 41, 43,
v	Sumpling coefficient	85, 206, 208, 226,
		231-233
С	Half-length of cuboid magnet, $\hat{z}$ direction	60, 62, 65, 67, 68
C C	Cube magnet side length	91, 103, 106
L	Cube magnet side tengen	91, 103, 100

Symbol	Description	Page
Ca	Damping coefficient of the vibration neutraliser	16, 18
d	Distance vector between the centres of two magnets, $\mathbf{d} = [d_x, d_y, d_z]^{T}$	xv, 60, 62, 63, 65–67,
$\mathbf{d}_1$	Position vector for differential magnet volume in cylindrical coordinates $\mathbf{d}_1 = [r_1, \phi_1, z_1]^T$	73 xv, 158
<b>d</b> <sub>2</sub>	Position vector for differential coil volume in cylindrical coordinates, $\mathbf{d}_2 = [r_2, \phi_2, z_2]^{T}$	xv, 158
d	Inclined magnets horizontal offset at the nominal position	108–112, 114,
d	Normalised magnet gap	117–120 xii, xiii, xix,
d $d_w$	Magnetic spring depth Diameter of wire	192–195, 197–211 103, 105, 106, 133 166–171, 173
e <sub>r</sub>	Radial distance between two cylindrical objects	164, 165
F	Force vector	57, 58, 68, 99, 109, 110, 116, 118, 119, 158
$\mathbf{F}_{z,x}$	Force between two cuboid magnets; the first magnetised in the $\hat{z}$ direction, the second in $\hat{x}$	66
$\mathbf{F}_{z,y}$	Force between two cuboid magnets; the first magnetised in the $\hat{z}$ direction, the second in $\hat{y}$	xxiii, 63, 66, 78
$\mathbf{F}_{z,z}$	Force between two cuboid magnets both magnetised in the $\hat{z}$ direction	xxiv, 61, 65, 66, 78, 80, 81, 109, 118
<b>F</b> <sub>eddy</sub> F	Eddy current force vector Magnitude of force	33 41, 68, 78, 80, 81, 85, 93, 102, 107, 110–113, 119, 120, 169–171, 173, 224
F(s) F <sub>c</sub>	Force input response, Laplace domain Axial force between a thick coil and a magnet	231, 232 156–159, 164, 167–169, 175, 176
F <sub>f</sub> F <sub>g</sub>	Axial force between two circular current loops Gravity force	156, 164 102
F <sub>r</sub>	Radial force between two cylindrical thin coils/magnets	78
$F_T$ $F_z$	Quasi-zero stiffness vertical magnet force function Axial force between two cylindrical thin coils/magnets	194, 206 76, 78, 157
F <sub>attr</sub> F <sub>repl</sub>	Magnet force (attraction magnet) Magnet force (repulsion magnet)	194 194

Symbol	Description	Page
$F_i$	Force characteristic of the inclined spring, with	xvi, 182, 186
	vertical component $F_{i_v}$ and horizontal component	
	$F_{i_h}$	
$F_v$	Force characteristic of the vertical spring, with	xvi, 182, 186
	vertical component $F_{v_v}$ and horizontal component	
	$F_{v_h}$	
$F_t$	Force characteristic of the overall inclined spring	xvi, 182–186
	quasi-zero stiffness mechanism, with vertical	
	component $F_{t_v}$ and horizontal component $F_{t_h}$	
$F_s$	Magnet force for coaxial cube magnets	64, 81, 193, 194, 196, 197
f	Force input	5-7, 231, 233
f <sub>a</sub>	Active force input for an inertial mass or vibration	viii, 16, 17
Ju	neutraliser	
$f_d$	Force disturbance	5
$f_q$	Normalised force at equilibrium	198, 199
$f_T$	Normalised Quasi-zero stiffness vertical magnet	194, 197–200, 206
	force	
$f_s$	Normalised magnet force for coaxial cube	64, 193–195
	magnets	
G	Force vector between inclined magnets in the local	109
	coordinate systems of the fixed magnet	,
G	Axial gap between the dual coils	175, 176, 219
G	Gap between magnets in various magnetic spring	91, 99, 100, 103, 133,
	designs	141, 142
G(s)	Transfer function	231–233
8	Acceleration due to gravity	85, 107, 111, 119,
		120, 179, 198, 200,
		206, 224
8	Normalised magnet gap	222, 226
8a	Feedback gain on relative acceleration	7-9, 11
8c	Feedback gain on relative velocity	7, 8, 10, 11
8d	Feedback gain on absolute acceleration	7–9, 11
8k	Feedback gain on relative displacement	7–9, 11, 85
8m	Feedback gain on absolute displacement	7-9, 11
$g_v$	Feedback gain on absolute velocity	7, 8, 10, 11, 232, 233
Н	Magnetic field strength vector	48–51
Н	Height of the apparatus	214, 215, 222–224
Η	Length of vertical spring under load	181, 183, 186, 189
$H_0$	Undeflected vertical spring length	180, 181, 183, 189
$H_c$	Coercivity	51, 52
h	Inclined spring vertical dimension	180–183, 186

Symbol	Description	Page
h	Multipole array height	133, 136–138, 140,
		141
h	Normalised nominal magnet displacement	192, 194, 195,
		197–199, 201, 206,
		210
$h_b$	Beam height	215, 223
$h_c$	Height of the coil	219
$h_m$	Height of the magnets support	214, 215, 222–224
$h_q$	Normalised displacement at equilibrium	198–201, 205, 206,
_		208
$h_s$	Height of the laser sensor	214, 215, 223
$h_{\epsilon}$	Height 'buffer' to account for additional	222, 224
	thicknesses	
T	Comment	
Ι	Current	48, 75, 155–159, 161,
		164, 167, 170, 171,
I	Indinad magnet system moment of inertia	175
I <sub>m</sub> i	Inclined magnet system moment of inertia The imaginary number $\sqrt{-1}$	119, 120 7, 8, 10, 226
i	Magnet index in the $\hat{x}$ direction.	-
L	Magnet fildex in the x direction.	144–147
J	Current density vector	48, 158
$\mathbf{J}_m$	Equivalent 'surface current' vector due to	48
<b>2</b>	magnetisation	
J <sub>eddy</sub>	Eddy current density vector	33
j	Magnet index in the $\hat{y}$ direction.	144–147
$\mathbf{K}_{z,y}$	Stiffness between two cuboid magnets; the first	63
	magnetised in the $\hat{z}$ direction, the second in $\hat{x}$	
$\mathbf{K}_{z,z}$	Stiffness between two cuboid magnets magnetised	xxiii, 62
	in the $\hat{z}$ direction	
Κ	Magnitude of stiffness characteristic (derivative of	87–90, 92
	force with respect to displacement)	
$K_c(s)$	Controller transfer function	232, 233
$K_h$	Stiffness characteristic in the horizontal direction	186, 187
$K_q$	Stiffness at equilibrium	200, 205, 208, 210
$K_T$	Quasi-zero stiffness vertical magnet stiffness	194, 200, 206, 208,
<b>.</b>	function	210
$K_v$	Stiffness characteristic in the vertical direction	184
K <sub>attr</sub>	Magnet stiffness (attraction magnet)	205
K <sub>repl</sub>	Magnet stiffness (repulsion magnet)	205
$K_s$	Magnet stiffness characteristic for coaxial cube	64, 193
	magnets	

Symbol	Description	Page
k	Stiffness coefficient	xxiii, xxiv, 5, 7–10,
		16, 18, 39, 41, 43,
		107, 110, 111, 113,
		120, 179, 224, 226,
		231–233
$k_i$	Inclined spring stiffness	180–184, 187
$k_v$	Vertical spring stiffness	180–183, 186
<i>k</i> <sub>a</sub>	Stiffness of the vibration neutraliser	16, 18
$k_T$	Normalised Quasi-zero stiffness vertical magnet stiffness	194, 197, 206
k <sub>lin</sub>	Linearised stiffness at a certain point	43
$k_s$	Normalised magnet stiffness characteristic for coaxial cube magnets	64, 65, 193, 194
L	Length of inclined spring under load	181, 182, 186, 189
$L_0$	Undeflected inclined spring length	180–182, 186, 189
$L_b$	Beam length	215
$L_c^{\nu}$	Coil length	75, 154, 155,
-	0	157–159, 161, 164,
		166, 167
$L_m$	Magnet length	82–84, 154, 155,
		157–159, 161, 164,
		167, 172, 215, 219,
		222–224
1	Displacement vector between magnet centre and	118
	centre of mass (lever arm)	
1	Lever arm	103, 105, 117–120
1	Multipole array length	133–136, 138
1	Normalised nominal magnet displacement	64, 65, 193, 195
$l_m$	Horizontal offset of the magnets support	214, 215, 223
$l_s$	Horizontal offset of the laser sensor	214, 215, 223
$l_w$	Length of the wire in the coil	166, 167
М	Magnetisation vector	28, 48–50, 57, 144,
	0	158
Ŵ	Unit magnetisation vector	144-147
$M_{\rm sat}$	Magnetisation at saturation	50, 51
m	Magnetic dipole	48
т	Mass	xxiii, xxiv, 5, 7–10,
		16, 18, 41, 43, 85,
		107, 119, 120, 179,
		198–202, 204–209,
		226, 231–233
m <sub>a</sub>	Vibration neutraliser mass	16, 18

<b>Symbol</b> <i>m</i> <sub>eq</sub>	<b>Description</b> Equivalent mass	<b>Page</b> 111, 224
N N N	Magnet grade, units MG Oe Multipole array number of magnets Number of turns in the coil	55 133–136, 144, 146 75, 155, 158, 159,
$N_m$	Magnet equivalent 'turns' for filament current model	164, 175 156, 157, 161
Nr	Number of turns in the radial direction	155–157, 160, 161, 166, 167, 175
$N_z$	Number of turns in the axial direction	155–157, 161, 166, 167
<b>î</b> n	Surface normal vector Exponential for empirical magnet force equation	57, 158 195–197, 200
$P_{bb}(\omega)$	Power spectrum accelerometer measurements of the base disturbance	39, 40, 228
$P_{mb}(\omega)$	Cross spectrum accelerometer measurements between the moving magnet and base disturbance	39, 40
$P_{mm}(\omega)$	Power spectrum accelerometer measurements of the moving magnet	228
p	Displacement vector due to rotation around centre of mass	118
р	Disturbance	41, 43, 85, 86
р	Dual-multipole array horizontal offset	150, 151
$p_b$	Beam pin origin height	214, 215, 223
$p_g$	Magnet gap	222, 223
$p_m$	Low magnet height	214, 215, 222, 224
$p_n$ $p_q$	High magnet height Quasi-zero stiffness position	214, 215, 222–224 222, 223
$\begin{array}{c} Q_{0} \\ Q_{1} \\ Q_{2} \\ Q_{3} \\ q_{0}(d) \\ q_{2}(d) \\ q_{4}(d) \end{array}$	Coefficient for empirical magnet force equation Coefficient for empirical magnet force equation Coefficient for empirical magnet force equation Coefficient for empirical magnet force equation Polynomial coefficient for modelling magnet force Polynomial coefficient for modelling magnet force Polynomial coefficient for modelling magnet force	195–197, 200 195–197, 200 195 195 194, 195 194, 195 194, 195
$R_x  R_y  R_z  R$	Planar rotation matrix around the $\hat{x}$ axis Planar rotation matrix around the $\hat{y}$ axis Planar rotation matrix around the $\hat{z}$ axis Distance vector between a magnet's centre and one of its corners/nodes (floating magnet)	66, 67 66 66, 109, 118 62, 63, 73

Symbol	Description	Page
R	Multipole array number of magnets per	133–139
	wavelength	
R	Coil resistance	166–171, 173, 218,
		219
$R_c$	Thick coil outer radius	155, 157–159, 161,
		164, 166, 167, 175,
_		219
$R_m$	Magnet outer radius	82–84, 154–159, 161,
		164, 166, 167, 172,
		215, 219
r	Distance vector between a magnet's centre and	62, 63
	one of its corners/nodes (fixed magnet)	
r	Euclidean distance of $\delta$ , $\sqrt{\delta_x^2 + \delta_y^2 + \delta_z^2}$	62–64
r	Radial component of distance vector in cylindrical	xv, 158, 163, 164
	coordinates	
r	Radius	156, 164, 165
r <sub>c</sub>	Thick coil inner radius	154, 155, 157–159,
		161, 164, 166, 167,
		175, 219
rg	Clearance between magnet and inner coil radii	154, 167
S	Vector of magnet side lengths (floating)	65–67
S	Integration surface	57, 158
$S_f$	Factor of safety	171, 173
$S_w$	White noise variance	41-43
s	Position vector	109, 118
s	Vector of magnet side lengths (fixed)	65–67
S	Differential region of the integration surface	57, 158
S	Laplace variable	xv–xvii, xx, xxi, 7, 8,
		231–233
$\mathbf{T}_{z,y}$	Torque between two cuboid magnets; the first	78
~,9	magnetised in the $\hat{z}$ direction, the second in $\hat{y}$	,
$\mathbf{T}_{z,z}$	Torque between two cuboid magnets magnetised	xxiv, 73, 78
-/-	in the $\hat{z}$ direction	
Т	Period	40
$T(s), T(\omega)$	Transmissibility	xxi, 8, 10, 39, 226,
., .,	·	228, 230, 231
$T_{\rm RSS}$	Root-sum-square of the transmissibility	10, 230, 231
	magnitude	
$T_z$	Inclined magnets torque	118, 119
t	Displacement vectors from the spring magnet	118
	centres to the centre of rotation in the coordinate	
	system of the magnets	

<b>Symbol</b> t	<b>Description</b> Time	<b>Page</b> 7, 40, 41, 43, 208,
$t_b$	Beam shell thickness	231 215, 223
U u	Potential energy Cuboid magnet unit length, cube root of volume	28 110–112, 114, 117, 119, 120
V	Volume	33, 48, 80, 81, 83, 110, 114, 158,
$V(\omega)$ <b>v</b> v	Variance gain, alternative of transmissibility <i>T</i> Velocity vector Differential region of the integration volume	167–171, 173 40 33 33, 158
W w w <sub>b</sub>	Multipole array number of wavelengths Inclined spring horizontal dimension Beam width	133–140 180–182, 186 215
$X_1(s)  X_2(s)  x  x  \hat{x}$	Base response, Laplace domain Vibration mass response, Laplace domain Displacement vector Horizontal displacement of the inclined spring Cartesian unit vector	7, 8, 18, 231–233 7, 8, 18, 231–233 57–59 180–187, 189 x, xiv, xvi–xviii, xx, xxiii, 60, 61, 66, 67, 69, 78, 88, 91, 92, 94–99, 101–103, 105, 106, 109, 118, 121, 124, 144–146, 149, 150
x	Displacement	41, 43, 80, 81, 179, 222, 223
x	Inclined magnets horizontal displacement	,, 109, 110, 112, 118, 119, 121–123
<i>x</i> <sub>1</sub>	Base displacement	viii, 5, 7, 8, 16, 40, 192, 206, 208, 220, 221, 231, 233
<i>x</i> <sub>2</sub>	Displacement of the vibration mass	viii, 5, 7, 8, 16, 40, 192, 194, 206, 220, 221, 231, 233
$\begin{array}{c} x_a \\ x_b \end{array}$	Vibration neutraliser displacement Displacement of the beam at the position of the laser sensor from 'zero'	16 223
$x_m$	Displacement of the magnets support	223, 224

Symbol	Description	Page
$x_p$	Projected displacement of the beam to the	223
	magnets support	
$x_s$	Displacement measured by the laser sensor	214, 215, 222, 223
$x_m$	Magnet centre position	224
Ŷ	Cartesian unit vector	x, xiv, xvi, xvii, xx,
		xxiii, 60, 61, 63, 66,
		68, 70, 71, 86, 90,
		94–98, 103–105, 107,
		109, 121, 123, 124,
		144–146, 149–151
у	Horizontal displacement	85, 86, 149, 150
y	Inclined magnets vertical displacement	108–111, 113,
		118–123
ź	Cartesian unit vector	x, xiv, xvi, xviii, xx,
		xxiii, xxiv, 60, 61,
		63, 66, 68, 70, 71, 86,
		88, 91–99, 101–107,
		109, 117, 118,
		144–146, 149, 150
Z	Axial displacement	155–159, 164, 165,
		167–169, 175, 176
Z	Vertical displacement	85, 86, 99, 100, 103,
		136, 140, 141, 149,
		150, 180–186, 189,
		217
Z	Axial component of distance vector in cylindrical coordinates	xv, 158, 164
Z	Inclined magnets out-of-plane displacement	109, 110, 118
$z_{\min}$	Maximum deflection of the vertical spring	183
α	Ratio between the inclined and vertical spring	180, 181, 183, 184,
	stiffnesses	187–190
α	Cylindrical magnet aspect ratio	82–84, 154, 155,
		167–170, 173
β	Coil aspect ratio	154, 155, 167–170,
		173
γ	Ratio between the inclined spring width and height	xxii, 182–187, 189
$\gamma$	Ratio between magnet lengths in a quasi-Halbach array	138, 140
$\gamma$	Square-face cuboid magnet aspect ratio	80, 81, 84, 110, 120
δ	Distance vector between a pair of corners/nodes	xx, xxii, 61–63, 73
	of two magnets, $\boldsymbol{\delta} = \begin{bmatrix} \delta_x, \delta_y, \delta_z \end{bmatrix}^{T}$	

Symbol $\delta$	<b>Description</b> Maximum displacement bound	<b>Page</b> 199, 201–203,
$\delta \\ \epsilon \\ \epsilon$	Displacement increment Percentage difference between $\gamma$ and $\gamma_{QZS}$ Closest (normalised) allowable displacement from quasi-zero stiffness to avoid instability	205–207 110, 111, 113 185, 186 199, 201–203, 205, 207
ζ η	Damping ratio, $0.5c/\sqrt{km}$ . Ratio between inclined and vertical spring lengths	xxiv, 120, 208, 226, 232, 236 180, 181, 183, 184,
$\eta_k$ heta	Nonlinearity measure Magnet rotation/inclination	186–190 206–208, 210
v	Multipole array magnetisation rotation between	67–70, 72, 108–112, 114, 117–120 133–135, 146
$\vartheta_0$	successive magnets Magnetisation direction of the first magnet in a multipole array	133, 146
$\vartheta_{xz}$	Magnetisation direction in the $\hat{x}$ - $\hat{z}$ plane of the first magnet in a multipole array	146
$\vartheta_{yz}$	Magnetisation direction in the $\hat{y}$ - $\hat{z}$ plane of the first magnet in a multipole array	146
$\kappa_{z,y}$	'Stiffness' between two magnetic nodes for cuboid magnets magnetised in the $\hat{z}$ and $\hat{y}$ directions respectively, used to calculate $\mathbf{K}_{z,z}$	63, 64
$\kappa_{z,z}$	'Stiffness' between two magnetic nodes for cuboid magnets magnetised in the $\hat{z}$ direction, used to calculate $\mathbf{K}_{z,z}$	62, 63
κ	Stiffness ratio	205
λ	Multipole array wavelength	133–136, 139
μ	Magnetic permeability of a material	30, 31, 49, 52
$\mu_0$	Magnetic permeability of the vacuum	48–52, 55, 57, 58, 61–65, 68, 73, 75, 76, 156–158, 164, 274
$\mu_r$	Relative permeability of a material	49, 50
ν	Ratio of magnet length squared to face area	84
ξ	Inclined spring normalised displacement in the	182–185, 187, 188,
0	load bearing direction Resistivity	190 166 16 <del>7</del>
$ ho \sigma$	Conductivity	166, 167 33
Φ	Magnetic flux vector	55 52
<b>ф</b> <i>z</i> ,у	'Force' between two magnetic nodes for orthogonally-magnetised cuboid magnets, used to calculate $\mathbf{F}_{z,y}$	63

Symbol	Description	Page
$oldsymbol{\phi}_{z,z}$	'Force' between two magnetic nodes for cuboid magnets magnetised in the $\hat{z}$ direction, used to calculate $\mathbf{F}_{z,z}$	61, 62, 73
$\phi$	Angular component of distance vector in cylindrical coordinates	xv, 158, 163, 164
$\varphi$	Inclined magnets planar rotation	116–123
ψ	'Torque' between two magnetic nodes for cuboid magnets magnetised in the $\hat{z}$ direction, used to calculate $T_{z,z}$	73
ω	Frequency	xix–xxi, 7, 8, 10, 39, 40, 202, 204, 205, 226, 228, 230–232
$\omega_d$	Resonance frequency, $\omega_n \sqrt{1-\zeta^2}$	200, 206, 207
$\omega_n$	Natural frequency, $\sqrt{k/m}$	xxiv, 107, 111, 179, 208, 224–226